

# Coefficient of Performance Computation of a Lecture Hall

Abhinav Misra<sup>1\*</sup>, Ashish Katiyar<sup>2</sup>, Hirendra Singh<sup>3</sup>

<sup>1</sup>P.G. Scholar, Department of Mechanical Engineering, Naraina College of Engineering and Technology, Kanpur, India <sup>2,3</sup>Assistant Professor, Department of Mechanical Engineering, Naraina College of Engineering and

Technology, Kanpur, India

Abstract: With improved living standards and rising ambient temperatures, human comfort is the most significant metric in the current environment. Electric air conditioning equipment is not recommended for large structures since it consumes a lot of electricity and has a short lifespan. Central air conditioning delivers greater dependability as well as fewer maintenance costs. Centralized air conditioning systems are installed in large buildings including commercial complexes, auditoriums, and office towers. With the increase in the students enrolled year after year, research and teaching institutions also demand comfort ability. The successful design of a refrigeration unit reduces electricity usage, lowers capital costs, and aids in the improvement the premises aesthetically. The research work illustrates about load of refrigeration evaluation in a climatic circumstance using the CLTD method. The MS-Excel tool can simply express the calculation of refrigeration load from things like occupancy gain of heat, illumination gain of heat, penetration and aeration gain of heat. The application is able to compute the refrigeration load from walls and roofs. The estimated figures were compared to the standard statistics compiled by the ASHRAE Fundamental Hand Books, and the findings were found to be validated.

*Keywords*: cooling load, central air conditioning, ton of refrigeration, CLTD, human comfort, HVAC.

## 1. Introduction

At the moment, there is an ecological equilibrium problem associated with energy use by enterprises and structures. Architecture, industry, commercial properties, residential dwellings, and marketplaces absorb 72 percent of global energy. In a big air - conditioned structure or complex, approximately energy of about 60 percent full amount needed for a structure assigned to the atmosphere habituation system install for cooling purposes. For reducing power consumption, we must supply a precise quantity of refrigeration and thermal load, suitable sized HVAC unit, also appropriate HVAC system monitor. External climatic parameters such as ambient temperature, sun radiation, and dampness are the primary elements that influence cooling demand. Native climate patterns are also significant variables for energy efficient buildings since energy usage in buildings is affected by climate patterns, plus effectiveness HVAC unit occurs like a result. Buildings would be more comfortable and energy efficient as a consequence.

Accurate cool and heat stack forecast, correct size for HVAC unit, plus optimized operation for unit are critical for minimizing power usage. Undeviating sun emission, transmitted stack, aeration/permeation stack plus interior stack are the constituents for premises refrigeration load. Computing and combining all of these loads yields an approximation of overall refrigeration stack. Computed load in this manner is the total sensible load. Based on building structure, a specific percentage from it will be usually added to account for dormant stack.

Principles for thermal transport and sun emissions are used to estimate load. CLTD, heat factors for the sun gain, and factors for refrigeration load computed for design circumstances and building materials utilized. Cosmic radiation computation principles used for estimating building's undeviating and deviating sun heat gain components. When mostly the constituents are put together, the overall load of chilling (or thermal) within the structure is calculated.

The potential loads are mentioned in the load sheet formats. Direct radiation from Sun, transmitting stack via exterior wall of varying degrees of lagging, enclosures, every conceivable wall type, ceiling, floor, plus outside atmosphere stack are among them. All transmission loads, along with ventilation/infiltration loads, are presented as a factor of the temperature differential between outside and interior air in this load calculation formulation.

The thermal load computation allows us to implement precise air conditioners equipment and air handling units. It also improves human comfort and air movement in the airconditioned vicinity.

The CLTD approach is used in this research to estimate the cooling load of a Lecture Hall

## 2. Effective Temperature

Human beings' perception of hotness and coldness is mostly based upon below mentioned parameters:

• DBT (Temperature- Dry bulb)

<sup>\*</sup>Corresponding author: abhinavm148@gmail.com

- RH (Humidity- Relative) and
- Velocity of Air

Cumulative sum involving three impacts is known as effective temperature, which also is described as the indicator that co-relates the cumulative effect on humans based on temperature of air, humidness plus velocity of air.

The chart of Human comfortability is a realistic implementation about notion for efficient temperature, which is shown in Fig 1.1. This figure is a study of ASHRAE research involving several types of persons treated to a wide variety of ambient temperature, relative humidity, and air velocity.



## 3. Heat Transfer Scrutiny

Heat is transported through numerous mediums in any structure or area, including exterior walls, top roof, ground floor, windows, and doors. Heat transport occurs through three modes: radiation, convection, as well as conduction. The refrigeration load for a structure determined by various factors including local climate, material, thermal properties and building type. There is other software available for load of refrigeration estimations, like HAP 4.3, Elite, BLAST as well as DOE 2.1E. Such software employs TFA plus HBA. Because such approaches need sophisticated as well as extensive entry of data, the majority of consultants' don't apply such approaches.

Better choose simpler and extra packed in technique to this application. The one-step process provided in the 2019 ASHRAE Handbook of Fundamentals is a more simple technique for determining load of refrigeration via approach of transfer function. This approach also known as cooling load temperature differences (CLTD) technique and it uses hands-on calculations for computation of refrigeration load.

Manual measurements are performed within enclosed premises (Lecture Hall) utilizing all of the equations, calculation processes, and material described. In order to determine thermal load, all of the equations above must be used about movement of heat in the course of the premises plus the interior stack. The computations then entered into a specific application called MS Excel to achieve the study objectives.

The following are the typical bit by bit processes in

estimation of overall load of heat:

- Determine the interior premises situation (Temp., RH).
- Choose exterior premises conditions (Temp., RH).
- Calculate overall coefficient of heat transfer (U<sub>o</sub>) of each parameters of premises.
- Determine the square units for each window, door, rooftop &floor.
- Determine transmitted gain of heat.
- Determine the gain of heat from sun.
- Determine rational as well as dormant gain of heat due to aeration, permeation, plus occupant.
- Determine the illumination gain of heat.
- Determine overall gain of heat, at last
- Determine Tonnes of Refrigeration.

#### 4. Design Condition

The quantity of refrigeration needed to make location pleasant in the summer and winter is determined by the intended inside conditions as well as the external temperatures on any particular day.

Such circumstances are considered as "Design of Indoor Environment" and the "Design of Outdoor Environment" respectively.

The following are the recommended interior temperatures and relative humidity levels for human comfort.

 Table 1

 Recommended interior temperatures and relative humidity levels for human comfort

Temperature-Dry Bulb (DBT)		Humidity-Relative (RH)		
Summer	22.7°C - 26.1°C	Summer	50 %	
Winter	22.1°C - 22.2 °C	Winter	20-30%	

The refrigeration load of Lecture Hall is based on a dry bulb temperature (DBT) of 23 degrees Celsius and a relative humidity (RH) of 50 percent, this combination of Temperature & relative humidity depicting the scenario of indoor design.

The outside design circumstances are based on public data for the given area, which is based on weather bureau or aviation statistics. Kanpur City's suitable climatic circumstances are 41.2 °C DBT and 45 percent relative humidity (month of May).

The refrigeration load computation for the Lecture hall is given in an MS EXCEL worksheet.

The cumulative statistical worksheet represents heat acquisition from a various parameters of heat transmission and the required load of refrigeration for cooling purpose.

PERFO	TCIENT OF ORMANCE TATION OF A URE HALL	Climatic Condition	DBT	WBT	RH (%)
	ge Of Engineering ology (NCET)	Outside	41.2	21.5	45
		Inside	24		50
Y MAY	(2021-22)	Diff.	17.2		
SUMMER		No. of Air Chan	ge ner	Hour =	1
Widths(mtrs)	Elevation (mtrs)	Total Region( sq.mtrs)	Total Cubic Region (mtrs cube)		
5.67	4 36	53			
	BPF = 0.12				·
			1	n	
Area (mtr sq.)					
2.4	0.14	96.14	0	32.30	
	0.12	144.58		0	
	0.15	131.5		0	
			-		
	0.53	131.57		0	
	0.48	96.14		0	
ITE & BOOT	SOLAD S TRANS	MISSION PE LT CLER			
				n	7
Area (mtr sq.)					
8.93		23.7		226.45	5587
	1.07	25.7			
			_		
-					
	1.07	23.7			
& TRANSMISSIO	ON HEAT GAIN EX	CLUDING WALLS &	ROOF	7	
Area (mtr sq.)		Temp. Diff. ( <sup>0</sup> C)	-	W	
			-	320.8128 2439.856339	
HEAT GAI	N THROUGH INF	ILTERATION			
Bypass	Factor	Temp. Diff. ( <sup>0</sup> C)		W	6
1	20.44	17.2	1	1364.0	8384
INTER	NAL SENSIDI F H	FAT CAIN			
LNIEK		LAI GAIN	1	T	
60					
15	15		0	22	
				1.1	
OOM SENSIBI E	HEAT CAIN	0411.2	64717		
		CURER L ROOM			
Bypass	- LATENT HEAT Factor	Diff. (kg/kg)		W	7
1	50000	0.0144	-	187	
					Č.
	Factor			N	0
60				W 270	0
60	Factor				10
60	Factor				10
	Factor 45				
ROOM LATENT	Factor 45 HEAT GAIN ctor (5%)	457 228 4800	.86		
ROOM LATENT 1	Factor           45           HEAT GAIN           ctor (5%)           OM (RLH)	457	.86 5.06	270	
ROOM LATENT 1 Loss & Safety Fac HEAT OF A RO IN A ROOM (RT	Factor           45           HEAT GAIN           ctor (5%)           OM (RLH)	457 228 480 14687.	.86 5.06	270	
COOM LATENT (Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible 1	Factor 45 HEAT GAIN tor (5%) OM (RLH) H) = RSH+RLH ION - OUTSIDE A Heat Gain By Outside	457 228 4800 14687. IR HEAT GAIN Air (OASH)	.86 5.06	270	
OOM LATENT Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible BPF	Factor 45 HEAT GAIN tor (5%) OM (RLH) H) = RSH+RLH HON - OUTSIDE A Heat Gain By Outside Factor	457 228 4800 14687. IR HEAT GAIN Air (OASH) Temp. Diff. ( <sup>0</sup> C)	.86 5.06	270	ŕ
COOM LATENT (Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible 1	Factor 45 HEAT GAIN tor (5%) OM (RLH) H) = RSH+RLH ION - OUTSIDE A Heat Gain By Outside	457 228 4800 14687. IR HEAT GAIN Air (OASH)	.86 5.06	270	ŕ
ROOM LATENT Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible 1 BPF 0.88	Factor 45 HEAT GAIN tor (5%) OM (RLH) H) = RSH+RLH HON - OUTSIDE A Heat Gain By Outside Factor	457 228 4800 14687. IR HEAT GAIN Air (OASH) Temp. Diff. ( <sup>6</sup> C) 17.2	.86 5.06	270	ŕ
COOM LATENT I Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible I BPF 0.88 Latent H BPF	Factor 45 HEAT GAIN tor (5%) OM (RLB) H) = RSH+RLH ION - OUTSIDE A Heat Gain By Outside Factor 20.44 eat Gain By Outside J Factor	457 228 480 14687. IR HEAT GAIN Air (OASH) Temp. Diff. ( <sup>6</sup> C) 17.2 Air (OALH) Diff. (kg kg)	.86 5.06	270 W 773.4	496
ROOM LATENT I Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible I BPF 0.88 Latent H	Factor 45 HEAT GAIN tor (5%) OM (RLH) H) = RSH+RLH Heat Gain By Outside Factor 20.44 eat Gain By Outside	457 228 4800 14687. IR HEAT GAIN Air (OASH) Temp. Diff. <sup>(°</sup> C) 17.2 Air (OALH)	.86 5.06	270 W 773.4	496
COOM LATENT I Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible 1 BPF 0.88 Latent H BPF 0.88	Factor 45 HEAT GAIN tor (5%) OM (RLB) H) = RSH+RLH ION - OUTSIDE A Heat Gain By Outside Factor 20.44 eat Gain By Outside J Factor	457 228 4800 14687. IR HEAT GAIN Air (OASH) Temp. Diff. <sup>(</sup> C) 17.2 Air (OALH) Diff. (kg/kg) 0.0144	.86 5.06 99295	270 W 773.4 W 158	496
ROOM LATENT I Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible I BPF 0.88 Latent H BPF 0.88 UB TOTAL :	Factor 45 HEAT GAIN stor (5%) OM (RLH) H) = RSH+RLH Heat Gain By Outside Factor 20.44 eat Gain By Outside Factor 50000	457 228 480 14687. IR HEAT GAIN Air (OASH) Temp. Diff. ( <sup>6</sup> C) 17.2 Air (OALH) Diff. (kg/kg) 0.0144 17045.	.86 5.06 99295 44255	270 W 773.4 W 158	496
COM LATENT Closs & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible 1 BPF 0.88 Latent H BPF 0.88 UB TOTAL : closs & Safety Fac	Factor 45 HEAT GAIN stor (5%) OM (RLH) H) = RSH+RLH Heat Gain By Outside Factor 20.44 eat Gain By Outside Factor 50000	457 228 480 14687. IR HEAT GAIN Air (OASH) Temp. Diff. ( <sup>6</sup> C) 17.2 Air (OALH) Diff. (kg/kg) 0.0144 17045. 852.27	.86 5.06 99295 44255 21276	270 W 773.4 W 158	496
ROOM LATENT I Loss & Safety Fac HEAT OF A RO IN A ROOM (RT CALCULAT Sensible I BPF 0.88 Latent H BPF 0.88 UB TOTAL :	Factor 45 HEAT GAIN stor (5%) OM (RLH) H) = RSH+RLH Heat Gain By Outside Factor 20.44 eat Gain By Outside Factor 50000	457 228 480 14687. IR HEAT GAIN Air (OASH) Temp. Diff. ( <sup>6</sup> C) 17.2 Air (OALH) Diff. (kg/kg) 0.0144 17045.	.86 5.06 99295 44255 21276	270 W 773.4 W 158	496
	& Techn       KANPURU U (26.475 !       Y     MAY       SU       SU       Solar s       Area (mtr sq.)       2.4       Area (mtr sq.)       Area (mtr sq.)       Area (mtr sq.)       Area (mtr sq.)       RAPE       Area (mtr sq.)       2.4       3.53       1       Area (mtr sq.)       2.4       53       1       Area (mtr sq.)       2.4       53       0.001 SENSIBLE       0.001 SENSIBLE       0.001 SENSIBLE       0.001 SENSIBLE       CALCULATION	& Technology (NCET)           KANPUR/UTTAR PRADESH (26.475 N & 80.225 E)           Y         MAY (2021-22)           SUMMER           Widths(mtrs)         Elevation (mtrs)           5.67         4.36           SDLAR HEAT GAIN THRO           Area (mt sq.)         Factor           0.49         0.12           0.15         0.23           0.53         0.48           0.15         0.33           0.48         0.17           8 rea (mt sq.)         Factor (W/m <sup>2</sup> . <sup>0</sup> C)           Area (mt sq.)         Factor (W/m <sup>2</sup> . <sup>0</sup> C)           1.07         1.07           8.93         1.07           8 transmission HEAT GAIN EX           Area (mt sq.)         Factor (W/m <sup>2</sup> . <sup>0</sup> C)           2.4         6.16           53         2.48           53         4.51           HEAT GAIN THROUGH INF           Bypass         Factor           60         70           15         15           000M SENSIBLE HEAT GAIN           000M SENSIBLE HEAT GAIN           CALCULATION - LATENT HEAT	& Technology (NCET)         Outside           KANPUR/UTTAR PRADESH (26.475 N & 80.225 E)         Inside           Y         MAY (2021-22)         Diff.           SUMMER         No. of Air Chan           Widths(mtrs)         Elevation (mtrs)         Total Region( sq.mtrs)           5.67         4.36         53           BPF = 0.12         SOLAR HEAT GAIN THROUGH GLASS           Area (mtr sq.)         Factor         Wind           0.49         43.24           2.4         0.14         96.14           0.15         131.57           0.53         131.57           0.53         144.58           0.48         96.14           0.48         96.14           0.48         96.14           1.07         23.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7           1.07         25.7	& Technology (NCET)         Outside         11.2           KANPUR, UTTAR PRADESH (26.475 N & 80.225 E)         Inside         24           Y         MAY (2021-22)         Diff.         17.2           SUMMER         No. of Air Change per         Inside         24           Widths(mtrs)         Elevation (mtrs)         Total Region( sq.mtrs)         Total Region	& Technology (NCET)         Outside         41.2         21.5           KANPUR, UTTAR PRADESH (26.475 N & 80.225 E)         Inside         24         24           Y         MAY (2021-22)         Diff.         17.2           SUMMER         No. of Air Change per Hour =         0.6         0.7           Widths(ntrs)         Elevation (mtrs)         Total Region(sq.mtrs)         Total Cubi (mtrs of 23           SUMMER         No. of Air Change per Hour =         0.15         23           BFF = 0.12         SUAR HEAT GAIN THROUGH GLASS         Area (mtr sq.)         Factor         W/m2         W           Area (mtr sq.)         Factor         W/m2         W         0.15         131.5         0           0.15         131.5         0         0.23         79.38         0           0.15         131.5         0         0.48         96.14         20.24           0.15         131.5         0         0.7         1.07         20.7           Area (mtr sq.)         Factor (Wm <sup>2</sup> ·C)         Temp. Diff. /C)         W           Area (mtr sq.)         Factor (Wm <sup>2</sup> ·C)         Temp. Diff. /C)         W           1.07         25.7         1         1.07         25.7

Fig. 2. Load estimation worksheet

### 5. Result and Outcome

This section gives the conversation of the after effects of created worksheet for refrigeration load computation of a

Lecture Hall for solace cooling in a structure of Research and Training Institute. The refrigeration load computation worksheet has been independently evolved dependent on measurements readings observed and required standard values sourced from ASHRAE Fundamental Data Book for Kanpur (UP, India) 26°N latitude. A portion of the analysts and HVAC specialists compute the refrigeration load by thumb rule. In the current investigation, refrigeration load estimation worksheet has been created from ASHRAE standard and followed CLTD strategy and refrigeration load can be obtained precisely.

The optimum refrigeration loads, sensible heat proportions, latent heat proportions of the Lecture Hall with a seating of 60 all at once of Institutional structure has been determined utilizing cooling load temperature contrast (CLTD) strategy.

The detailed result including assessment of maximum Refrigeration Load required for Air conditioning of Lecture Hall, also mention as COP estimation of the Lecture Hall was carried out, and the below mentioned following statistical results have been obtained:

HEAT TRANSMISSION (W)	HEAT GAINS IN A ROOM (W)	HEAT GAIN FROM OUTSIDE	V infiltration (m3/min)	
Through Glass	Sensible Heat Gain	Sensible Heat Gain	3.88	
320.8128	9866.18	773.44		
Through Ceiling	Latent Heat Gain	Latent Heat Gain		
2439.85	4806.06	1584		
Through Floor	Total Heat Gain	otal Heat Gain Total Heat Gain		
602.85	14672.24	2357.44		
Maximum Total Heat Gain (W) =		17881.17		
Tonnes Of Refrigeration (TR) =	5.11 (TONS)			

Fig. 3. Result and Outcome

#### 6. Conclusion

In this research paper, the investigation of a lecture hall, which is part of a research and training institute located in Kanpur, was taken under consideration for computation of refrigeration loads. The sensible refrigeration difference of temperature (CLTD) technique applied for obtaining overall load of refrigeration i.e. the COP of a system during the month of May in summer.

From the calculations, investigation and after effects of this exploration work, the following inferences have been made:

- Software (MS-Excel) base Refrigeration load computation can be achieved efficiently, speedily plus with extra precision.
- In the current exploration, observed about refrigeration load depends on the ambient degrees. Like the surrounding degree lowers, chilling request lowers furthermore, vice-versa.
- The developed worksheet incorporated by some stage like:
  - 1. Immediate sunlight based radiation heat acquire for glass windows
  - 2. Transmission heat acquires

- 3. Inward warmth acquires
- 4. Ventilation and invasion
- 5. Absolute outcome and show person

Watt (W), Kilowatt (kW), BTU/h also Ton of refrigeration (TR) can be the units of refrigeration required for the system.

People, light, infiltration, and ventilation are parameters of cooling load items that may be simply inserted into the MS-Excel software.

All the parameters involved in the process of computation of refrigeration load are of utmost importance and put in correct order definitely gives the desired outcomes for any system of air conditioning.

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